Free Convection Fresnel Lenses Concentrator

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Abstract — This project involves an evaluation of improved Fresnel lenses solar energy concentrator. Its innovation consists of preventing forced convection heat losses at the receiver area. Heat losses by convection and by the transmittance and reflectance properties of the solar rays transmission of the material were calculated for the comparison of parabolic and Fresnel lenses solar energy concentrators. The expected efficiency of the Fresnel lenses concentrator within free convection conditions is 93%. An efficiency increment to 95% is noticed for vacuum conditions. Doing the same analysis for the parabolic concentrator the efficiency expectation is 95% within forced convection conditions. The efficiency expectations for both concentrators presents them as competitive choices for energy production.

Key Terms — Free and Forced Convection, Reflectance, Thermal Efficiency, Transmittance.

INTRODUCTION

In a typical sunny day the earth receives an average amount of Direct Normal Irradiance (DNI) of 900 [1]. In year 2009, Abbot [2] said that with this irradiation most of the earth surface could be used to satisfy the current global energy consumption with leftovers.

Professor Carlos Alvarado, Ph. D. P.E. developed a Capstone Design Project with the interest to a further investigation project with the vision to create an efficient solar energy concentrator.

This Investigation has the objective to create an innovative design of solar energy concentrator that can compete with the efficiency of current popular concepts using low cost materials by minimizing the energy losses by forced convection.

CURRENT CONCEPTS FOR SOLAR ENERGY CONCENTRATORS

Solar energy concentrators consist of two main components for the heat flux collection on the receiver. See Figure 1. The losses from the sunlight due transmittance or reflectance of the material as soon as the light makes contact with the concentrator and the losses due convection.

There are seven types [3] of solar concentrators that are known as distinguished designs during the development of this technology. This types are known as:

- Parabolic Concentrator
- Hyperbolic Concentrator
- Fresnel Lens Concentrator
- Compound Parabolic Concentrator (CPC)
- Dielectric Totally Internally Reflecting Concentrator (DTIRC)
- Quantum Dot Concentrator (QDC)

According to the optical principles we can categorize each concentrator in four groups. See Table 1 below [3]:

Table 1
Four Different Groups of Concentrators

Group	Description	
Reflector	Upon hitting the concentrator, the sun rays will	
	be reflected to the receiver.	
	Example: Parabolic Trough, Parabolic Dish,	
	CPC Trough, Hyperboloid Concentrator.	
Refractor	Upon hitting the concentrator, the sun rays will	
	be refracted to the receiver.	
	Example: Fresnel Lens Concentrator	
Hybrid	Upon hitting the concentrator, the sun rays can	
	experience both reflection and refraction	
	before hitting to the receiver.	
	Example: DTIRC, Flat High Concentration	
	Devises	
Luminescent	The photons will experience total internal	
	reflection and guided to the receiver.	
	Example: QDC	

Each type of solar concentrator present their advantages and disadvantages on their years of effective service. See Table 2 below [4].

Table 2
Summary of the Advantage and Disadvantage of the Concentrators

Type of	Advantage	Disadvantage
Concentrator		
Parabolic	•High concentration	Requires larger
Concentrator		field of view.
		 Need a good
		tracking system.
Hyperboloid	• Compact	• Need to
Concentrator		introduce lens at
		the entrance
		aperture to work
		effectively.
Fresnel	Thinner than	Imperfection
Concentrator	conventional lens.	on the edges of
lens	• Requires less	the facets,
	material than	causing the rays
	conventional lens.	improperly
	Able to separate the	focused at the
	direct and diffuse light	receiver.
	- suitable to control the	
	illumination and	
	temperature of a	
	building interior.	
Compound	Higher gain when its	Need a good
Parabolic	field of view is	tracking system.
Concentrator	narrow.	
Dielectric	Higher gain than	• Cannot
Totally	CPC.	efficiently
Internally	 Smaller sizes than 	transfer all of the
Reflecting	CPC.	solar energy that
Concentrator		it collects into a
		lower index
		media.
Flat High	Compact.	Difficulty to
Concentration	 Very high 	create electrical
Devices (RR,	concentration	connection and
XX, XR, RX,		heat sinking due
and RXI)		to the position of
		the cell.
		• The cell
		dimension must
		be designed to a
		minimum to
		reduce
		shadowing
		effect.
Quantum Dot	•No tracking needed.	• Restricted in
Concentrator	 Fully utilize both 	terms of
	direct and diffuse solar	Development
		-

radiation	due to the
	requirements on
	the luminescent
	dyes.

PROPOSAL OF IMPROVEMENTS TO SOLAR ENERGY CONCENTRATORS

The idea consist in developing a solar energy concentrator system with a Hybrid type technology based in both types Reflector and Refractor. Using mirrors placed on a structure with an angle to reflect the sunrays to the same area where the receiver is located. See Figure 1. Fresnel Lenses technology will take place with the purpose of refracting the sunrays to direct them to the receiver of reduced area. In order to avoid energy losses forced convection will be eliminated from the receiver area but natural convection will still take place on the investigation. However manufacturing for the Fresnel lenses solar energy concentrator could also use a vacuum chamber inside the structure to completely eliminate energy losses due convection.

SYSTEM CONCEPT DEVELOPMENT

The system is form of a cubical structure in which five of its six faces will have a fixed identical Fresnel lens unit for each side north, south, west, east and top. The face of the cubical structure that does not have a Fresnel lens is in which the structure will rest. Each Fresnel lens has the same properties. See Figure 1 below.

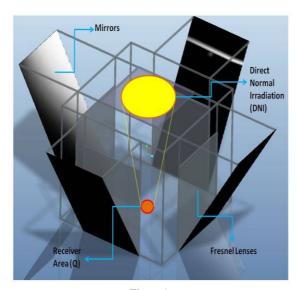


Figure 1
Fixed Cubical Structure

EQUATIONS AND MATHEMATICAL DESCRIPTION

There are two different analysis for the efficiencies of the Fresnel lenses concentrator. The first analysis is based on the efficiency of the receiver where the properties of the element as the absorber of DNI are considered. The device that is used in high temperature solar concentrators for the conversion of concentrated solar radiation to heat is called "receiver". It is designed to absorb the concentrated solar radiation and to transfer as much energy as possible to a heat transfer fluid. Losses originate from the fact, that the absorbing surface may not be completely black, that it emits thermal radiation to the environment, because it has an elevated temperature, and that convection as well as conduction occur. The heat entrance will be evaluated into a reduced capitation area with no concentration factor assuming that the DNI is orthogonal from the Fresnel lens surface and that the receiver is not protected by a transparent cover the heat entrance from DNI to the receiver can be calculated as (1)

$$Q_{solar} = A_a \cdot I \cdot \eta_{optic} \tag{1}$$

Where A_a is the aperture area of the Fresnel lens, I is the radiation density of the direct solar radiation or DNI and η_{optic} as the optic efficiency of the concentrator material in terms of reflectance Γ and transmittance τ respectively. The useful heat collected is denoted by the product of (1) and the absorptance properties of the receivers material yields as (2)

$$Q_{absorbed} = \alpha \cdot Q_{solar} \tag{2}$$

where α is the absortivity coefficient [5] of the receiver material. The heat losses will take place in the reradiating area with emissivity applying the Stefan Boltzmann Law and heat losses due convection as (3) [6].

$$Q_{lost} = A_r \cdot \left[\varepsilon \cdot \sigma \cdot T_r^4 - U(T_r - T_s) \right]$$
 (3)

where A_r is the receiver area, \mathcal{E} is the emissivity coefficient [7] of the absorber, σ stands for the Stefan Boltzmann constant, U_L is the convection coefficient [8][9], T_r is the temperature on the receiver and T_s is the temperature on the surroundings. Using (1), (2) and (3) the thermal efficiency for the receiver $\eta_{Receiver}$ is defined by the Carnot efficiency [10] as the ratio of the absorbed heat minus the heat losses and the heat entrance as (4)

$$\eta_{\text{Received}} = \frac{Q_{absorbed} - Q_{lost}}{Q_{solar}} \tag{4}$$

If a vacuum chamber takes place into the system the energy losses due convection are zero in (3). Therefore the heat losses Q_{lost} [10] will be denoted as (5)

$$Q_{lost} = A_r \cdot \left[\varepsilon \cdot \sigma \cdot T_A^4 \right] \tag{5}$$

However in this investigation DNI its concentrated to the receiver when its covered with fresnel lenses as a transparent cover of PMMA and energy losses occur due the transmittance of the

material. As for a parabolic concentrator energy losses from DNI occur due the reflectance properties of the material. Here is where the discussion for the second analysis for efficiency takes place on the concentrator itself where the property of transmittance or reflectance of the material of the concentrator is considered.

Parabolic concentrators uses mirrors to reflect the sunlight to a receiver. These mirrors are made with high reflective materials. AccuCoat inc. [11] demonstrates some material reflectivity. For aluminum a reflectivity of 88% to 93% and silver with a reflectivity of 95% to 97%. See Figure 2 and Figure 3 below [11].

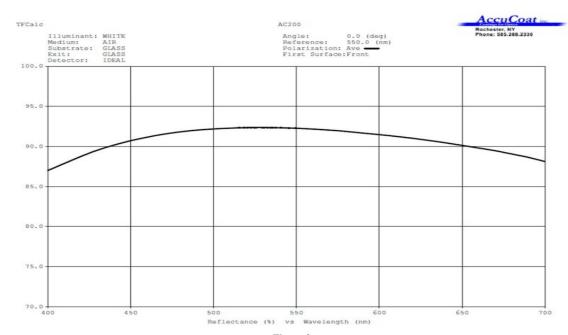


Figure 2
Protected Aluminum Coating Reflectance

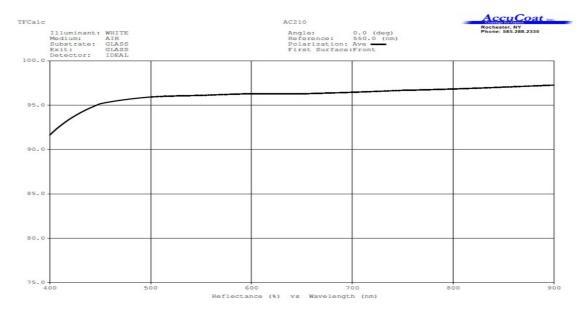


Figure 3
Protected Silver Coating Reflectance

However Fresnel lenses concentrators uses high transmittance materials such as polymer Plexiglass. Also known as Polymethyl Methacrylate or PMMA is a transparent thermoplastic material often used as a light weight

alternative to glass. It's also an economical alternative to Polycarbonate when extreme durability is not required. It's low cost, has great clarity. PMMA transmits T up to 93% of light. See Figure 4 below [12].

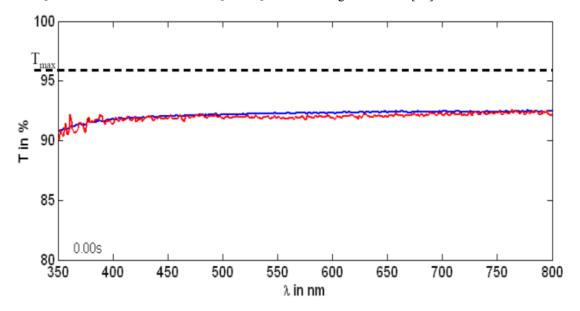


Figure 4
PMMA Transmittance Monitoring

RESULTS

The thermal analysis was executed for the Fresnel lenses concentrator and for the parabolic concentrator as well for comparison purposes. Tables 3 to 5 demonstrates the properties and working conditions for the receiver and for each solar concentrator respectively. Tables 6 to 10 demonstrates the resultant values for the thermal analysis and the efficiencies. An assumption for the Fresnel lenses concentrator is that only one of the five lenses is considered for the DNI due that the mirrors for the simultaneous solar capitation of the five lenses are not yet installed. No reflectance properties is considered for the PMMA as well as no transmittance for the aluminum and silver coatings.

Table 3
Common Values

Variable	Value	Units	Description
α	0.30		Absortivity Coeff.
			(Aluminum
			Polished) [9]
I	900	W/m^2	Direct Normal
			Irradiation (DNI) [1]
ε	0.77		Emissivity Coeff.
			(Aluminum
			Polished) [7]
$\overline{\sigma}$	5.67*10 ⁻	W/m ² K ⁴	Stefan Boltzmann
	8		Const. [8]
$U_{\it free}$	25	W/m ² K	Convection Coeff
free			Free Convection.
			(Air) [8][9]
$U_{\it forced}$	150	W/m ² K	Convection Coeff
forced			Forced Convection.
			(Air) [8][9]
T_r	573	K	Receiver
- r			Temperature
$T_{\rm s}$	300	K	Ambient
- s			Temperature

Table 4
Fresnel lens Properties

Variable	Value	Units	Description
A_a	0.79	m ²	Area of aperture
A_r	0.00065	m ²	Area of the receiver
τ	93%		PMMA Transmittance [12]

Table 5
Parabolic Properties

Variable	Value	Units	Description
A_a	5	m ²	Area of aperture
A_r	0.004	m ²	Area of the receiver
Γ_a	88%		Aluminum Reflectance [11]
Γ_s	95%		Silver Reflectance [11]

Table 6
Heat Entrance, Heat Absorbed and Heat Losses @ Free and
Forced Convection and Vacuum for PMMA Fresnel Lenses

Variable	Value	Units	Description
Q_{solar}	661	W	Fresnel lens heat
\mathcal{L}_{solar}			entrance @ free
			convection
Qahsorhed	198	W	Fresnel lens heat
∠absorbed			absorbed @ free
			convection
0	47	W	Fresnel lens heat
$Q_{lost,forced_conv}$			losses @ forced
			convection
0	7	W	Fresnel lens heat
$Q_{lost, free_conv}$			losses @ free
			convection
0.	3	W	Fresnel lens heat
∠lost,vacuum			lost @ vacuum

Table 7

Heat Entrance, Heat Absorbed and Heat Losses @ Forced
Convection for Parabolic Aluminum

Variable	Value	Units	Description
Q_{solar}	3960	W	Parabolic heat entrance
$Q_{absorbed}$	1188	W	Fresnel lens heat absorbed
$Q_{lost,forced}$	161	W	Fresnel lens heat losses @ forced convection

Table 8

Heat Entrance, Heat Absorbed and Heat Losses @ Forced
Convection for Parabolic Silver

Variable	Value	Units	Description
Q_{solar}	4275	W	Parabolic heat entrance @ free convection
$Q_{absorbed}$	1283	W	Fresnel lens heat absorbed @ free convection
$Q_{lost,forced}$	173	W	Fresnel lens heat losses @ forced convection

Table 9
Thermal Efficiencies for Fresnel Lens

Variable	%	Description
n	23	Fresnel lens absorber
$\eta_{abs,forced_PMMA}$		efficiency @ forced
		convection
n	29	Fresnel lens absorber
$\eta_{abs,free_PMMA}$		efficiency @ free
		convection
n	30	Fresnel lens absorber
$\eta_{abs_vac_PMMA}$		efficiency @ vacuum

Table 10
Thermal Efficiencies for Parabolic Aluminum and Silver

Variable	%	Description
$\eta_{abs,forced_alum}$	23	Parabolic absorber efficiency @ forced convection
$\eta_{abs,forced_silver}$	29	Parabolic absorber efficiency @ forced convection

WORK FOLLOW-UP

- Correctly calibrate the solar tracker and the linear actuators.
- Finish installation of thermocouples in the area of the entrance to the temperature readings.
- Installation of four mirrors for the collection of samples from the side lenses.
- Take real-time sampling and develop analysis and conclusion and of samples taken.
- Continue the research for efficiency increment on solar energy concentrators and their mechanisms.

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